AMENDMENTS TO THE SPECIFICATION

Please replace the paragraph at page 6, lines 8-22 with the following paragraph:

C1

FIG. 5 is a top view of microactuator 32 with the rotor in a displaced position. Upon actuation of microactuator 32, a force is generated to move magnet 44, thereby bending beam structure 46 and moving the rotor with respect to stator 38. Beam structure 46, and more particularly second beam pair 50, allows sufficient flexibility for the proximal end of the rotor (magnet bonding pad 42) to move in the direction indicated by arrows 45 when microactuator 32 is actuated and in the direction indicated by arrows 63 when the rotor is pulled away from the stator (as seen in FIG. 6). Flexibility in beam structure 46 is required for side to side movement of the rotor during actuation of microactuator 32. When the rotor finally positions slider 12 over a track of a disc it rotates side to side and slightly traces an arc 64 around rotation center 52. Prior art beam structures allowed a vertical deflection of approximately 100 microns. Beam structure 46 of the present invention prevents the ~~stator~~ rotor from significantly shifting vertically out of the horizontal plane of microactuator 32 and minimizes the amount of vertical deflection.

Please replace the paragraph at page 9, lines 5-22 with the following paragraph:

C2

Beam structure 46 operatively connects the rotor of microactuator 32 to stator 38 and prevents excessive movement and twisting of the stator out of the horizontal plane of microactuator 32. Beam structure 46 is comprised of first beam pair element 48 aligned with the width of the rotor and second beam pair element 50 in a dog-leg configuration and aligned with the length and the width of the rotor. During a head slap event, the increased thickness of beam elements 48 and 50 and the arrangement of beam structure 46 around the sides of the rotor, reduces the stress in the beam elements, prevents failure of the beam elements and keeps the ~~stator~~ rotor from significant movement out of the horizontal plane of microactuator 32. The rotor of the present invention is balanced about rotation center 52 defined by first beam pair 48. During hard seek acceleration by VCM 18, near perfect balance of the rotor about rotation center 52 results in significantly less stress induced in beam structure 46 by uncontrolled rotor shifting, thus decreasing the likelihood of fatigue stress in beam structure 46. During stiction loading of microactuator 32, deflection limiters 64 and 66 prevent slider 12 from being completely pulled out of stator 38. The inability for the rotor of the present

C2 cont'd invention to maintain its position within the horizontal and vertical planes of microactuator 32 is an advantage over prior art microactuators.